Belt Tension Calculator

First, the Effective Belt Tension (TE) must be calculated. TE is the sum of the tension required to move the empty belt (TC), the tension required to move the load horizontally (TL), and the tension required to lift the load (TH).

Example 1. TE = TC + TL + TH

$\mathbf{TC} = \mathbf{F}_1 \mathbf{x} \mathbf{L} \mathbf{x}$ CW	 F₁ = .035" [Normal friction factor for average conditions (over 20°F) to move empty belt.] L = Belt length (ft.) CW = Weight of conveyor belt components. (Table A of the Engineering Manual)
$\mathbf{TL} = \mathbf{F}_2 \mathbf{x} \mathbf{L} \mathbf{x}$ MW	$\mathbf{F}_2 = .04$ " [Normal friction factor to move load horizontally.] $\mathbf{L} = \text{Belt length (ft.)}$ $\mathbf{MW} = \text{Material weight (lbs. per lineal foot).}$ $\mathbf{MW} = 33.3 \text{ TPH } \mathbf{or}$ Total material load in lbs. $\overline{\text{Belt Speed (fpm)}}$
$TH = H \times MW$	H = Difference in elevation of terminal pulleys (ft.) $MW = Material weight (lbs. per lineal foot).$ $MW = 33.3 TPH or Total material load in lbs.$ $HW = 33.3 TPH or Total material load in lbs.$
$\mathbf{TE} = \mathbf{TC} + \mathbf{TL} + \mathbf{TH}$	See all calculations above. Continue with calculation of TS and then TO

Calculations

Second, additional tension must be added to the effective Belt Tension (Te) to prevent slippage on the drive pulley. This is called Slack Side Tension (Ts), or commonly called T2.

* Ts = D X Te

* D = Drive factor (Table B in Engineering Handbook: Drive Factor)

Next, the total tension To = Te + Ts

To is the total operating tension, or commonly called Allowable Working Tension, and is expressed in PIW (pounds per inch of width). To is sometimes referred to as TL.

Additional tension must be added to the Effective Belt Tension to prevent belt slippage on the drive pulley. This is called Slack Side Tension (TS).

TS = D x TE (Table B in Engineering Handbook: Drive Factor)

Total tension herein called Operating Tension (TO), sometimes called **Allowable Working Tension**, is the value used to select the reinforcement ply combinations.

The next step is to select the proper reinforcement (usually fabric) combinations. A number of alternatives are usually available, and the final selection is narrowed by the following considerations:

- 1. **Flexibility** to negotiate the pulleys. The minimum recommended pulley diameters for each fabric -- the number of plies combination must be considered at the percent of rated tension of the combination. If this is disregarded, service life will be shortened by ply delamination, splice failure, and tracking problems. (Table C, page 11-5 of the Engineering Manual).
- 2. **Troughability**. Eliminate combinations that will not allow normal belt contact with carrying idlers, including transition areas, while belt is empty. A belt with inadequate transverse flexibility will result in poor training and tracking which will cause edge wear, spillage, poor loading, and early belt failure. (Tables D1 and D2 on page 11-6 of the Engineering Manual).
- 3. **Impact Resistance.** The final selection must be rated to handle the lump weights and the weight per lineal foot of the material. Loading conditions will vary and must be considered. Your selection must be rated for the condition or the belt will be beaten into premature failure. TTables E1, E2, and E3 page 11-8 of the Engineering Manual).
- 4. **Load Support.** The final selection must be capable of resisting the load weight at the juncture of the center support idler and the troughing idlers. The belt carcass will break down and split longitudinally in this area if it is loaded beyond its capacity to support the load. (Table F1 page 11-10 of the Engineering Manual).

The values of the belt carcass to pass the above consideration should be available in the conveyor belt manufacturer reference manuals or printed information for belt selection.

The final selection of the alternate combinations should be made by considering costs and availability from a reliable source.